



# Spatiotemporal variations in ozone and carbon dioxide concentrations in an HVAC system of a LEED-certified office building

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## ABSTRACT

Indoor air quality (IAQ) is crucial for the health, well-being, and productivity of office occupants. IAQ is strongly influenced by occupancy and the operational mode of the heating, ventilation, and air conditioning (HVAC) system. This study investigates the spatiotemporal variations in ozone (O<sub>3</sub>) and carbon dioxide (CO<sub>2</sub>) concentrations throughout the HVAC system of a LEED-certified office building. A four-month field measurement campaign was conducted at the Ray W. Herrick Laboratories, employing an automated multi-point sampling system to monitor O<sub>3</sub> and CO<sub>2</sub> at eight locations throughout the HVAC system. The objectives of this study are to characterize the spatiotemporal distribution of these gases under different ventilation modes and occupancy levels, and to identify O<sub>3</sub> loss mechanisms in the office and its HVAC system. Spatiotemporal variations in O<sub>3</sub> and CO<sub>2</sub> concentrations were observed throughout the HVAC system. Results indicate that outdoor air exchange rates (AERs) significantly impact indoor O<sub>3</sub> levels, with higher AERs resulting in increased indoor O<sub>3</sub> but reduced CO<sub>2</sub> concentrations. Measurements reveal that HVAC filters and ducts contribute to O<sub>3</sub> loss, with up to 18% O<sub>3</sub> removal observed in the longest HVAC duct segment. Additionally, occupancy influences O<sub>3</sub> deposition onto human skin and clothing surfaces. This research underscores the limitations of ventilation standards that focus only on CO<sub>2</sub>, highlighting the need for ventilation strategies that consider the effects of occupancy and outdoor AERs on different gases. By integrating multi-point gas sampling into building automation systems, more effective control strategies can be developed to enhance IAQ and occupant health while reducing energy consumption.

## 1. Introduction

Indoor air quality (IAQ) in office buildings can impact the health, well-being, and productivity of the occupants [1–3]. Operation of building heating, ventilation, and air conditioning (HVAC) systems and occupant activities are two important factors that may affect the dynamics and chemistry of indoor air pollutants, including airborne particles, volatile organic compounds (VOCs), nitrogen oxides (NO<sub>x</sub>: NO and NO<sub>2</sub>), ozone (O<sub>3</sub>), and carbon dioxide (CO<sub>2</sub>). Among these indoor air pollutants, O<sub>3</sub> and CO<sub>2</sub> are two important trace gases that are strongly connected to IAQ. In most buildings, especially in office buildings without combustion sources, the exhaled breath of occupants is the major source of CO<sub>2</sub> [4,5]. O<sub>3</sub> is an important driver of indoor oxidative reactive chemistry. O<sub>3</sub> can react with VOCs such as monoterpenes emitted by human activities, including cleaning and disinfecting surfaces, and initiate the formation of secondary organic aerosol (SOA) [6–11]. In addition, O<sub>3</sub> can react with compounds on indoor surfaces

including building surfaces and human body surfaces. O<sub>3</sub> reacts with unsaturated compounds in human skin oil, including squalene, glycerides, fatty acids, and cholesterol, leading to the formation of volatile skin oil ozonolysis products and SOA [12–16]. O<sub>3</sub>-skin oil reactions are considered one of the major O<sub>3</sub> sinks in occupied indoor environments [12]. Both O<sub>3</sub> itself and its secondary reaction products can have adverse effects on human health. Human exposure to O<sub>3</sub> has been associated with respiratory and cardiovascular morbidity and mortality [17–21]. 4-oxopentanal (4-OPA), a reaction product of O<sub>3</sub> and skin oil, can cause irritation of the respiratory system and skin [22–25].

LEED-certified office buildings often implement sophisticated HVAC systems with building automation systems to control parameters related to the indoor thermal environment and IAQ, while minimizing energy consumption. Mixing ratios of indoor CO<sub>2</sub> are strongly associated with the design and operation of mechanical ventilation systems [26–29]. Outdoor and exhaust volumetric airflow rates determine the air exchange rate (AER) of the building, which directly impacts CO<sub>2</sub> loss and

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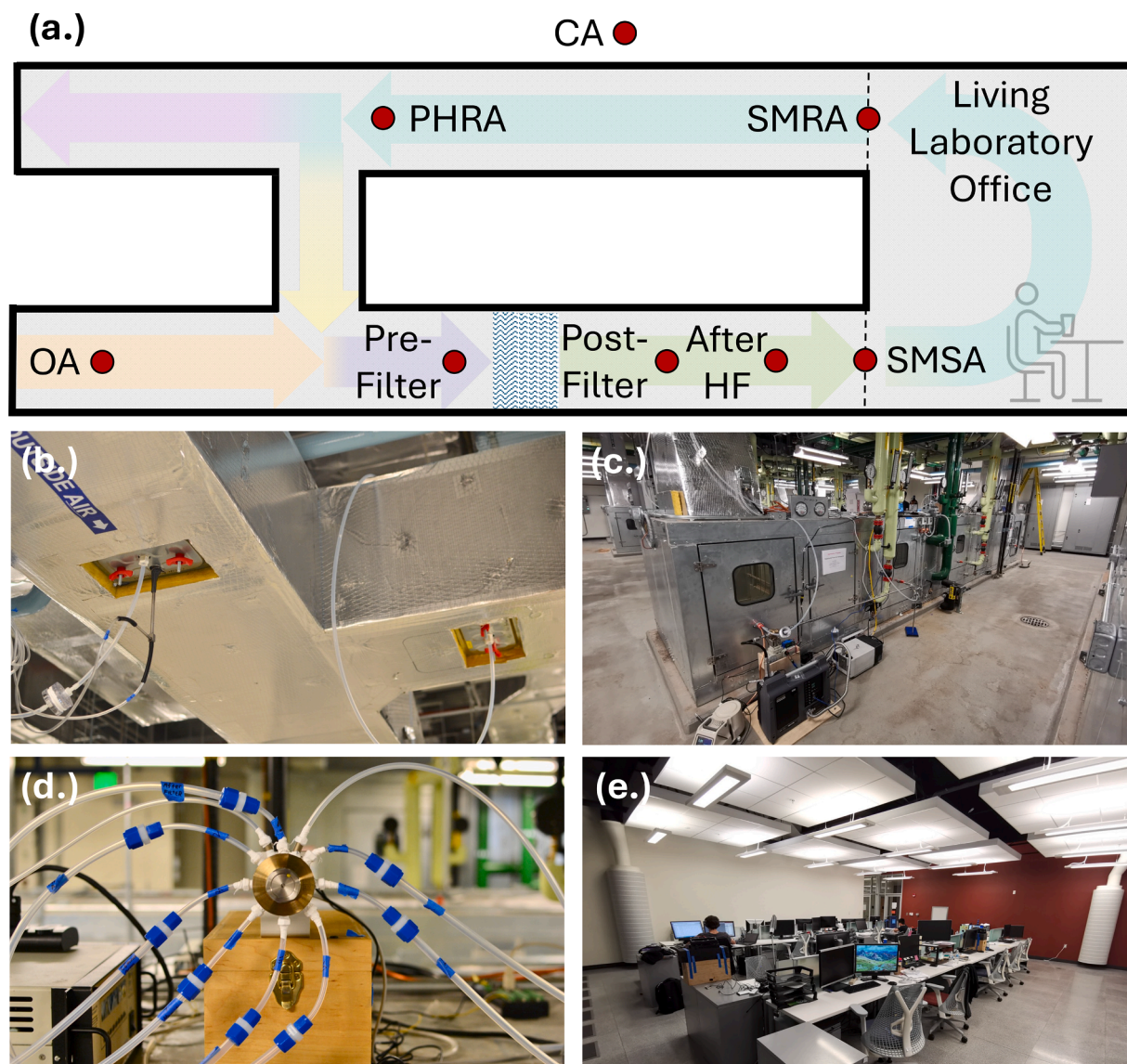
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**Fig. 1.** (a.) Schematic of the experimental setup for multi-point sampling of O<sub>3</sub> and CO<sub>2</sub> mixing ratios throughout the HVAC system of the living laboratory office. The eight sampling locations are noted in the schematic and include: the outdoor air intake (OA), upstream of the HVAC filter bank (Pre-Filter), downstream of the HVAC filter bank (Post-Filter), after the steam humidifier (After HF), supply air duct located in the small mechanical room (SMSA), return air duct located in the small mechanical room (SMRA), return air duct located in the penthouse (PHRA), and the common area (CA) adjacent to the office; (b.) photo of the OA and PHRA sampling locations in the penthouse; (c.) photo of the Pre-Filter and Post-Filter sampling locations from the HVAC system; (d.) photo of the programmable multi-flow path selector connected to the O<sub>3</sub> and CO<sub>2</sub> gas analyzers; and (e.) photo of the living laboratory office.

CO<sub>2</sub> mixing ratios in buildings. CO<sub>2</sub> mixing ratios are commonly monitored in building automation systems as an indicator of ventilation conditions, occupancy levels, or other indoor air pollutants [30–32]. ANSI/ASHRAE Standard 62.1–2022 has been established to regulate the minimum outdoor volumetric airflow rate in buildings based on the number of people present or the area of the office space [33]. However, most standards primarily focus on CO<sub>2</sub> levels and do not adequately address the complexities of IAQ, such as the presence and dynamics of other critical air pollutants like O<sub>3</sub>. This limitation can lead to ventilation strategies that fail to comprehensively protect occupant health and well-being.

A building's mechanical ventilation system may impact indoor O<sub>3</sub> mixing ratios and dynamics differently than that for CO<sub>2</sub>, which is an inert gas. Ventilation systems are the major pathway for outdoor O<sub>3</sub> to enter indoor environments [34–36], which could further impact indoor oxidative reactive chemistry and the formation of secondary gas- and particle-phase products. In addition, the surfaces of HVAC ducts and the

components of HVAC systems, including the HVAC filter bank and heating and cooling coils, could be a sink for outdoor O<sub>3</sub> as it is transported through the HVAC ducts before being supplied to the office [37–39]. However, current common HVAC system control strategies for IAQ are predominantly based on CO<sub>2</sub> sensing or modeling [40–43]. Furthermore, very few systems monitor trace gas concentrations at multiple locations within the HVAC system, failing to account for the impact of HVAC ducts and the operational conditions of HVAC components on IAQ in their control logic. Simultaneous real-time monitoring of the mixing ratios of these trace gases throughout building HVAC systems is critical to improve our understanding on indoor air pollutant dynamics and chemistry, providing data and information to determine appropriate ventilation strategies that improve IAQ.

However, to the authors' knowledge, there have been no prior long-term, multi-location measurements of O<sub>3</sub> and CO<sub>2</sub> throughout a commercial HVAC system under different ventilation and occupancy conditions. This study proposes and presents a novel automated multi-

location sampling system that has a wide range of applications, especially for ventilation control and improving IAQ. Specifically, this study investigates how the HVAC system operational mode and occupancy impact the spatiotemporal distribution of O<sub>3</sub> and CO<sub>2</sub> mixing ratios, as well as O<sub>3</sub> dynamics, in a ventilation system of a LEED-certified office building. A four-month field measurement campaign was conducted from February to June 2019 in one of the four occupied open-plan living laboratory offices at the Purdue University Ray W. Herrick Laboratories, which was awarded a LEED Gold Certificate. An automated multi-point sampling system was designed and built to monitor O<sub>3</sub> and CO<sub>2</sub> mixing ratios throughout the HVAC system in real-time. The objectives of this study are: (1.) to characterize the spatiotemporal distribution of O<sub>3</sub> and CO<sub>2</sub> mixing ratios at eight different sampling locations throughout the ventilation system of the office building under different HVAC system operational modes, including outdoor AERs and pressurization conditions, and different occupancy levels; (2.) to understand how the indoor/outdoor (I/O) ratio of O<sub>3</sub> and CO<sub>2</sub> mixing ratios are related to the operational mode of the mechanical ventilation system and human occupancy levels; and (3.) to understand loss mechanisms for O<sub>3</sub> in the office and throughout its HVAC system.

## 2. Materials and methods

### 2.1. Site description: Herrick living laboratory open-plan offices at Purdue University

The study site, one of the four Herrick living laboratory offices, is part of a LEED-certified building at Purdue University in West Lafayette, IN, U.S. (40°25'19.4"N 86°55'11.6"W). The living laboratories are four reconfigurable, side-by-side large open-plan office spaces, with a maximum occupancy of 20 and interior volume of 333 m<sup>3</sup>. Each office includes its own HVAC system, which is equipped with a MERV-8 pre-filter and a MERV-14 final-filter (Fig. 1). A building automation system (Niagara/AX, Tridium Inc., Richmond, VA, U.S.), along with hundreds of sensors, is applied to achieve real-time monitoring and precise control of the HVAC system. These sensors monitor air temperatures, relative humidities, volumetric airflow rates, damper positions, and fan speeds at different locations throughout the HVAC system; this provides a detailed airflow profile for the entire system. Damper positions and fan speeds were changed throughout the campaign to adjust the supply, return, and outdoor volumetric airflow rates.

### 2.2. Experimental setup and air quality instrumentation

#### 2.2.1. Automated multi-point sampling system for the monitoring of O<sub>3</sub> and CO<sub>2</sub> mixing ratios

An automated multi-point sampling system was built with a programmable multi-flow path selector (EUTA-2VLS8MWE2, Valco Instruments Co. Inc., Houston, TX, U.S.) to sample O<sub>3</sub> and CO<sub>2</sub>, as well as NO<sub>x</sub> and VOCs, at eight locations throughout the HVAC system (Fig. 1). The sampling locations included: the outdoor air intake (OA), upstream of the HVAC filter bank (Pre-Filter), downstream of the HVAC filter bank (Post-Filter), after the steam humidifier (After HF), supply air duct located in the small mechanical room (SMSA), return air duct located in the penthouse (PHRA), and the common area (CA) adjacent to the living laboratory office (Fig. 1). Perfluoroalkoxy (PFA) tubing (3/8" (9.5 mm) OD) was used for the sampling lines to connect each sampling location to the multi-flow path selector. The selected flow path outlet was connected to trace gas analyzers. The unselected streams were purged by a rough pump to prevent accumulation of stagnant air in the lines. This was done to eliminate a time delay in sampling. The total flow rate of the selected stream was maintained at 11.25 L min<sup>-1</sup> with a vacuum pump drawing the sample air at 9 L min<sup>-1</sup>. This ensured that the residence time of the sample air was no more than 8 s. A polytetrafluoroethylene (PTFE) membrane filter was installed at the inlet of each sampling line to

**Table 1**

Multi-point sampling sequences implemented during the field measurement campaign.

Sequence	Multi-Point Sampling Sequence
Spatial	4 min outdoor air (OA) + 4 min Pre-Filter + 4 min Post-Filter + 4 min after humidifier (After HF) + 4 min supply air (SMSA) + 4 min return air (SMRA) + 3 min return air (PHRA) + 3 min common area (CA)
Temporal	5 min outdoor air (OA) + 5 min supply air (SMSA) + 20 min return air (SMRA)
Pre-/Post-Filter	4 min Pre-Filter + 4 min Post-Filter

remove particles and was replaced daily to ensure its efficacy.

#### 2.2.2. Instrumentation for the real-time monitoring of O<sub>3</sub> and CO<sub>2</sub> mixing ratios and office occupancy

An O<sub>3</sub> gas analyzer (Model M400E, Teledyne Technologies Inc., Thousand Oaks, CA, U.S.) and a CO<sub>2</sub> gas analyzer (LI-830, LI-COR Biosciences, Lincoln, NE, U.S.) were connected to the selected flow path outlet of the multi-flow path selector to alternatively monitor O<sub>3</sub> and CO<sub>2</sub> mixing ratios at the eight locations in real-time (1 Hz). The CO<sub>2</sub> analyzer was operated at a flow rate of 0.75 L min<sup>-1</sup>. The O<sub>3</sub> gas analyzer monitored O<sub>3</sub> mixing ratios with a precision of 1 ppb using a photometric analyzer based on ultraviolet absorption of O<sub>3</sub> at 254 nm, with a sample flow rate at 0.8 L min<sup>-1</sup>. Nitrogen oxides (NO and NO<sub>2</sub>) were measured by a chemiluminescence NO-NO<sub>2</sub>-NO<sub>x</sub> analyzer (Model 42C, Thermo Electron Corp., Waltham, MA, U.S.) with a sample flow rate at 0.6 L min<sup>-1</sup>. VOCs with a proton affinity greater than water were monitored by a proton transfer reaction time-of-flight mass spectrometer (PTR-TOF-MS, PTR-TOF 4000, Ionicon Analytik Ges.m.b.H., Innsbruck, Austria) using H<sub>3</sub>O<sup>+</sup> as the reagent ion. The inlet flow rate was maintained at 0.1 L min<sup>-1</sup>. Details of the PTR-TOF-MS measurements can be found in Wu et al. (2021 and 2024) [44,45]. The O<sub>3</sub>, CO<sub>2</sub>, and NO-NO<sub>2</sub>-NO<sub>x</sub> analyzers were calibrated before the measurement campaign. The PTR-TOF-MS was calibrated daily following protocols described in previous studies [6,45–49]. Human occupancy in the living laboratory was tracked via chair-embedded thermocouples during the measurement campaign as described in Wagner et al. (2021) [50].

#### 2.2.3. Multi-point sampling system sequences

During the measurement campaign, three different sequences for the multi-point sampling system were implemented. (1.) Sequence 1: a 30 min spatial sequence. The spatial sequence was designed to monitor spatial variations in O<sub>3</sub> and CO<sub>2</sub> mixing ratios across the eight locations of the HVAC system, which can inform how different HVAC components impact O<sub>3</sub> chemistry. During the spatial sequence, the valve system switched to OA, Pre-Filter, Post-Filter, After HF, SMSA, and SMRA for 4 min each, then PHRA and CA for 3 min each. (2.) Sequence 2: a 30 min temporal sequence, which specifically focused on O<sub>3</sub> and CO<sub>2</sub> dynamics in the office. This enabled for a more detailed investigation into the impact of occupants and ventilation conditions on indoor O<sub>3</sub> and CO<sub>2</sub> mixing ratios. During the temporal sequence, SMRA was sampled for 20 min, and then SMSA and OA were sampled for 5 min each. (3.) Sequence 3: an 8 min pre-/post-filter sequence. To investigate the role of the HVAC filter bank on O<sub>3</sub> dynamics, a pre-/post-filter sequence was designed to sample only the pre-filter and post-filter locations with 4 min of sampling at Pre-Filter and 4 min of sampling at Post-Filter. Details of each valve sequence are provided in Table 1. The O<sub>3</sub> and CO<sub>2</sub> mixing ratios for each location presented in the results are the mean values during the 30-min sample sequence for sequences 1 and 2. The first minute and last minute of data at each sampling location was disregarded to avoid the impact of the valve switching. For the pre-/post-filter sequence, data when the valve was sampling from another location was interpolated to better compare the difference in O<sub>3</sub> mixing ratios upstream and downstream of the HVAC filter bank.



**Table 2**

Ventilation modes and their corresponding outdoor AERs and office pressurization conditions.

Ventilation Mode	Outdoor AER	Office Pressurization
1	Low	No
2	Medium	No
3	High	No
4	Low	Yes
5	Medium	Yes
6	High	Yes
7	Natural Ventilation	-

**2.2.4. HVAC system operational modes during the measurement campaign**

To investigate how the operation of the HVAC system impacts indoor O<sub>3</sub> and CO<sub>2</sub> dynamics, the supply, return, and outdoor volumetric airflow rates were adjusted to achieve different outdoor AERs and room pressurization conditions under different ventilation modes. The ratio of the recirculation airflow rate to the total supply airflow rate was adjusted from 0 to 1 to achieve different AERs. Throughout the campaign, based on the AERs and pressurization conditions, the ventilation modes can be categorized as follows: (1.) Mode 1: low AER (≤ 2.7 h<sup>-1</sup>), no pressurization; (2.) Mode 2: medium AER (2.7 h<sup>-1</sup> < AER ≤ 5.4 h<sup>-1</sup>), no pressurization; (3.) Mode 3: high AER (> 5.4 h<sup>-1</sup>), no pressurization; (4.) Mode 4: low AER, pressurization; (5.) Mode 5: medium AER, pressurization; (6.) Mode 6: high AER, pressurization; and (7.) Mode 7: natural ventilation with unfiltered outdoor air via a south-facing, full-size double-skin glass façade. Table 2 summarizes the ventilation modes implemented during the campaign. The corresponding dates for each ventilation mode are summarized in a calendar in Fig. 2.

**2.3. Material balance model to characterize O<sub>3</sub> loss terms in the office**

To quantify the dynamics of O<sub>3</sub> in the living laboratory office, a simplified material balance model was developed to estimate the overall loss rate for O<sub>3</sub> under different ventilation and occupancy conditions:

$$\frac{dC_i(t)}{dt} = p\lambda_{in}(t)C_o(t) + \frac{C_o(t)Q_o(t)}{V} - L(t)C_i(t) \tag{1}$$

where C<sub>i</sub>(t) is the indoor O<sub>3</sub> mixing ratio (ppb) sampled at SMRA; p is the penetration factor for O<sub>3</sub> (-); λ<sub>in</sub>(t) is the infiltration rate (h<sup>-1</sup>); C<sub>o</sub>(t) is the outdoor O<sub>3</sub> mixing ratio (ppb) sampled at OA; Q<sub>o</sub>(t) is the outdoor air volumetric airflow rate (m<sup>3</sup> h<sup>-1</sup>); V is the volume of the living laboratory office (m<sup>3</sup>); and L(t) is the overall O<sub>3</sub> loss rate (h<sup>-1</sup>). The office was treated as a completely mixed flow reactor (CMFR). There were no obvious indoor O<sub>3</sub> sources in the office, such as photocopiers or air purifiers, thus, an indoor emission term for O<sub>3</sub> is not included in the material balance model.

The time-dependent overall loss rate L(t) for O<sub>3</sub> can be estimated by rearranging Eq. (1) as follows:

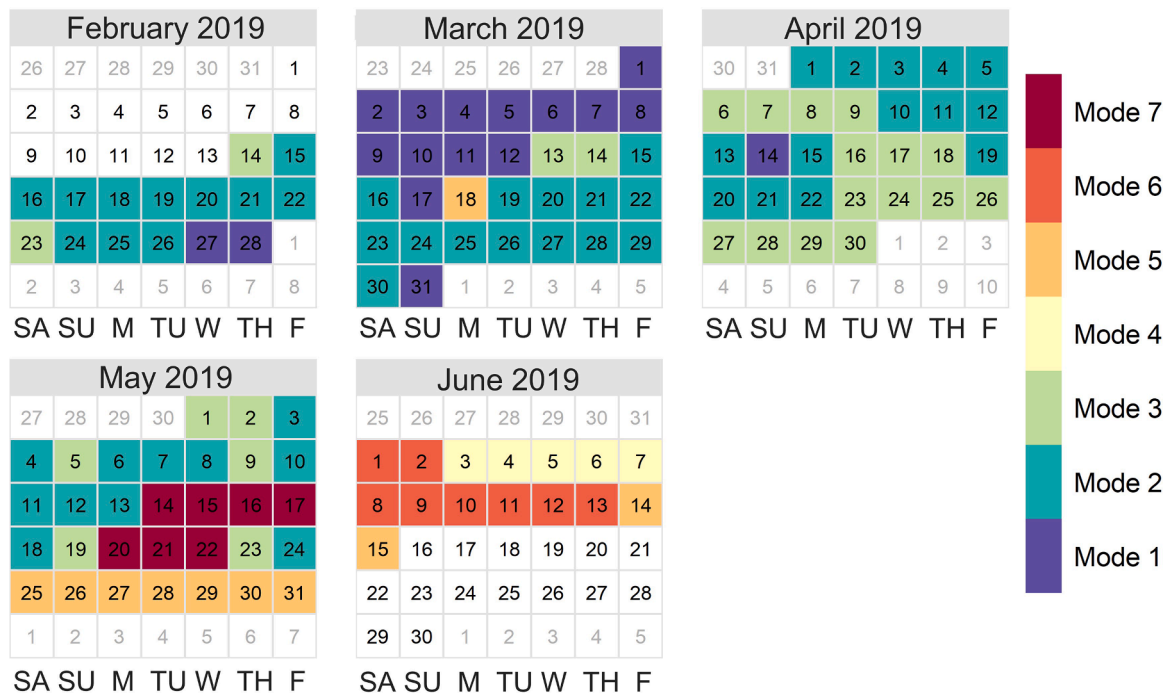
$$L(t) = -\frac{C_i(t+dt) - C_i(t)}{dt} - p\lambda_{in}(t)C_o(t) - \frac{C_o(t)Q_o(t)}{V} \tag{2}$$

For a 30 min spatial sequence or a 30 min temporal sequence, the time interval was defined as: dt = 30 min. The mean SMRA and OA O<sub>3</sub> mixing ratios during the 30 min period were taken as the values for C<sub>i</sub>(t) and C<sub>o</sub>(t), respectively. Similarly, the mean value of the outdoor air volumetric airflow rate during the 30 min period was taken as the value for Q<sub>o</sub>(t).

The overall loss rate for O<sub>3</sub> includes indoor O<sub>3</sub> removal by exfiltration, deposition to building surfaces and human body surfaces, building ventilation (outdoor air exchange), and gas-phase reactions with indoor NO<sub>x</sub> and selected VOCs, as shown in Eq. (3):

$$L(t) = \lambda_{ex}(t) + \frac{v_{d,bldg}S_{bldg}}{V} + \frac{v_{d,occ}(t)S_{occ}(t)}{V} + \frac{Q_o(t)}{V} + L_{reaction}(t) \tag{3}$$

where λ<sub>ex</sub>(t) is the exfiltration rate (h<sup>-1</sup>); v<sub>d,bldg</sub> is the O<sub>3</sub> deposition velocity to interior building material surfaces in the office (m h<sup>-1</sup>); S<sub>bldg</sub> is the surface area of interior building material surfaces in the office, assumed to be constant over time (m<sup>2</sup>); v<sub>d,occ</sub>(t) is the O<sub>3</sub> deposition velocity to office occupant surfaces (m h<sup>-1</sup>); S<sub>occ</sub>(t) is the surface area of



**Fig. 2.** Ventilation modes implemented throughout the four-month measurement campaign, including: (1.) Mode 1: low AER (≤ 2.7 h<sup>-1</sup>), no pressurization; (2.) Mode 2: medium AER (2.7 h<sup>-1</sup> < AER ≤ 5.4 h<sup>-1</sup>), no pressurization; (3.) Mode 3: high AER (> 5.4 h<sup>-1</sup>), no pressurization; (4.) Mode 4: low AER, pressurization; (5.) Mode 5: medium AER, pressurization; (6.) Mode 6: high AER, pressurization; and (7.) Mode 7: natural ventilation with unfiltered outdoor air.



**Table 3**  
Reaction rate constants of O<sub>3</sub> reactions with various gas-phase compounds.

Chemical Formula	Compound Name	Reaction Rate Constant (cm <sup>3</sup> molecule <sup>-1</sup> s <sup>-1</sup> )	Indoor Concentration Range (ppb)
C <sub>4</sub> H <sub>8</sub>	1-Butene	1.00E-17 [52]	0 – 10
C <sub>5</sub> H <sub>8</sub>	Isoprene	1.28E-17 [52]	0 – 2
C <sub>10</sub> H <sub>16</sub>	Limonene	2.08E-16 [51]	0 – 10
NO	Nitrogen monoxide	1.40E-12 [53]	0 – 1
NO <sub>2</sub>	Nitrogen dioxide	3.50E-17 [53]	0 – 2.5

the office occupants, which varies with the occupancy in the office (m<sup>2</sup>); and  $L_{reaction}(t)$  is the loss rate due to O<sub>3</sub> reactions with indoor NO<sub>x</sub> and selected VOCs, including butene (C<sub>4</sub>H<sub>8</sub>), isoprene (C<sub>5</sub>H<sub>8</sub>), and monoterpenes (C<sub>10</sub>H<sub>16</sub>) (h<sup>-1</sup>).  $L_{reaction}(t)$  can be expressed as:

$$L_{reaction}(t) = \sum_i k_i [M_i(t)] \quad (4)$$

where  $[M_i(t)]$  is the concentration of compound  $M_i$  (molecule cm<sup>-3</sup>) and  $k_i$  is the reaction rate constant of the reaction between O<sub>3</sub> and compound  $M_i$  (cm<sup>3</sup> molecule<sup>-1</sup> s<sup>-1</sup>). Limonene and isoprene, two VOCs associated with human-related emissions, and NO<sub>x</sub>, with outdoor air as the major source, were considered for O<sub>3</sub> reactions. 1-Butene, with relatively high concentrations observed during the campaign and a relatively high

reaction rate with O<sub>3</sub>, was considered as well. Their corresponding reaction rate constants and range of concentrations are summarized in Table 3. The reaction rate constants were obtained from the National Institute of Standards and Technology (NIST) Chemical Kinetics Database [51–54]. An average deposition velocity to interior building material surfaces in the office of  $v_{d,bldg} = 1.62 \text{ m h}^{-1}$  was applied throughout the loss rate calculation. With all other parameters known, the O<sub>3</sub> deposition velocity to human body surfaces,  $v_{d,occ}(t)$ , can be back calculated with the assumption that the surface area of each occupant is approximately 1.7 m<sup>2</sup> [55].

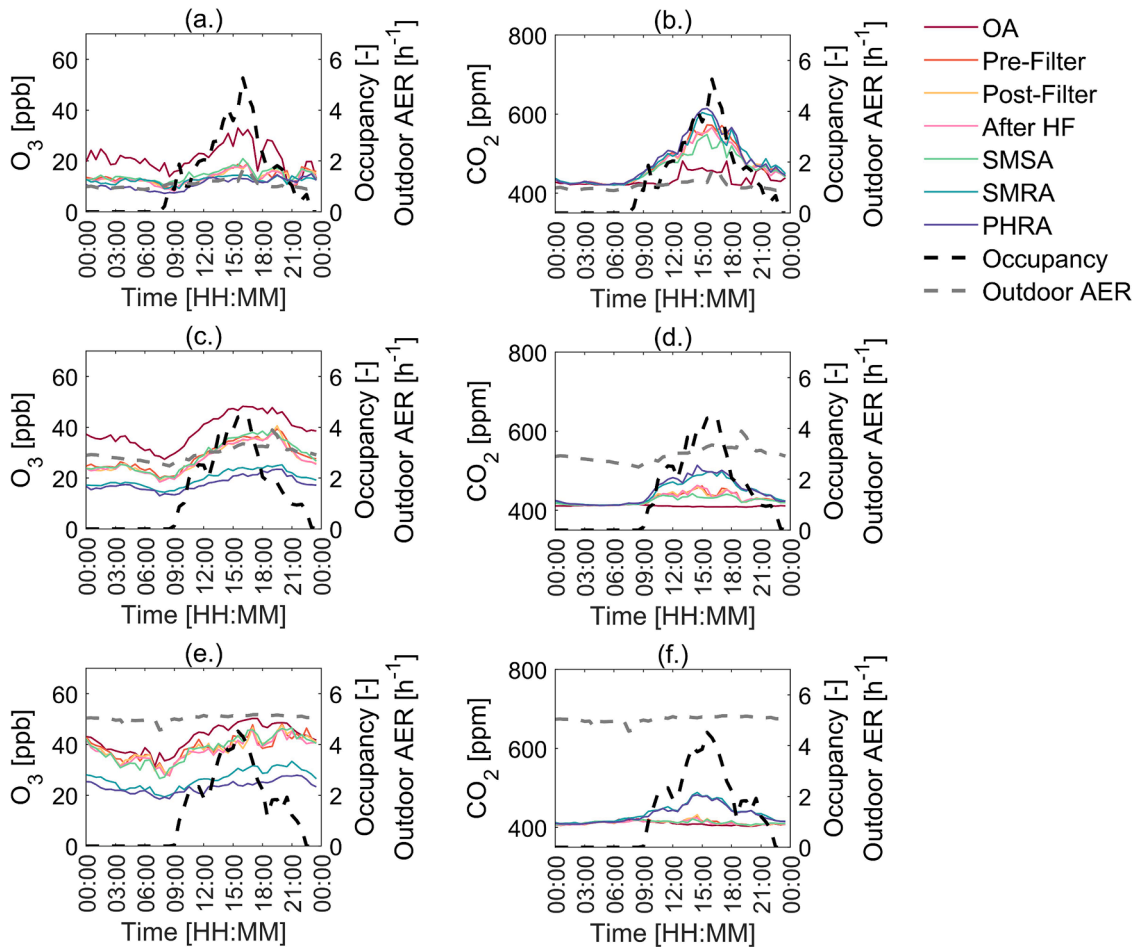
The loss mechanisms of indoor O<sub>3</sub> listed above contribute to the difference between indoor and outdoor O<sub>3</sub> mixing ratios, which can be described by the “O<sub>3</sub> loss” ( $C_{loss}(t)$ ). O<sub>3</sub> loss is a metric to evaluate the adverse health effects from human exposure to oxidation products from O<sub>3</sub> reactions and separate them from exposures to indoor O<sub>3</sub> itself [56]. The O<sub>3</sub> loss can be expressed as:

$$C_{loss}(t) = C_o(t) - C_i(t) \quad (5)$$

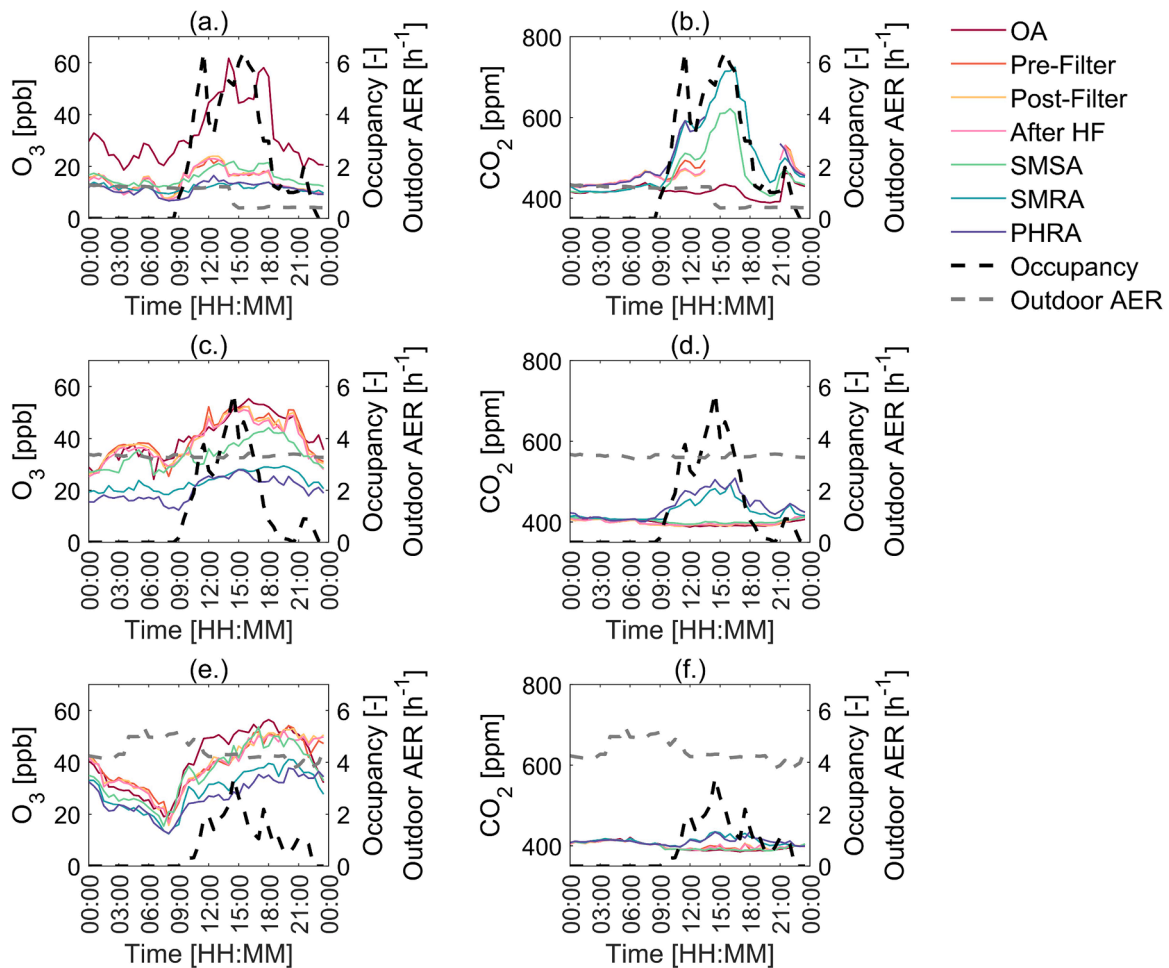
### 3. Results and discussion

#### 3.1. Spatiotemporal variations in O<sub>3</sub> and CO<sub>2</sub> mixing ratios throughout the HVAC system

The diurnal trends of O<sub>3</sub> and CO<sub>2</sub> mixing ratios at the seven sampling locations (CA not shown) throughout the HVAC system under the three



**Fig. 3.** Diurnal trends of (a.) O<sub>3</sub> and (b.) CO<sub>2</sub> mixing ratios under ventilation mode 1: low AER, no pressurization; (c.) O<sub>3</sub> and (d.) CO<sub>2</sub> mixing ratios under ventilation mode 2: medium AER, no pressurization; (e.) O<sub>3</sub> and (f.) CO<sub>2</sub> mixing ratios under ventilation mode 3: high AER, no pressurization. Occupancy (black dashed line) and outdoor AER (gray dashed line) are plotted on the right axis. The O<sub>3</sub> and CO<sub>2</sub> mixing ratios, outdoor AER, and occupancy are the median values of all days under the same ventilation mode category.



**Fig. 4.** Diurnal trends of (a.) O<sub>3</sub> and (b.) CO<sub>2</sub> mixing ratios under ventilation mode 4: low AER, pressurization; (c.) O<sub>3</sub> and (d.) CO<sub>2</sub> mixing ratios under ventilation mode 5: medium AER, pressurization; (e.) O<sub>3</sub> and (f.) CO<sub>2</sub> mixing ratios under ventilation mode 6: high AER, pressurization. Occupancy (black dashed line) and outdoor AER (gray dashed line) are plotted on the right axis. The O<sub>3</sub> and CO<sub>2</sub> mixing ratios, outdoor AER, and occupancy are the median values of all days under the same ventilation mode category.

non-pressurized ventilation operational modes 1 to 3 are presented in Fig. 3. O<sub>3</sub> mixing ratios at OA started increasing gradually at around 08:00 in the morning and peaked in the afternoon at around 17:00, which is consistent with previous observations of diurnal trends in outdoor O<sub>3</sub> mixing ratios [57–59]. Under ventilation modes 1 to 3 with low, medium, and high outdoor AERs, the SMRA O<sub>3</sub> mixing ratios peaked at 14, 25, and 33 ppb, respectively. As outdoor O<sub>3</sub> introduced via mechanical ventilation is the major source of indoor O<sub>3</sub> for this office, the outdoor AER has a strong impact on indoor O<sub>3</sub> mixing ratios. The SMRA CO<sub>2</sub> mixing ratio peaked at 15:00 to 16:00 in the afternoon, following the trend of the human occupancy level in the office. For example, as shown in Fig. 3(b), the occupancy level started increasing from 07:00 and reached its peak of 5.3 at around 16:00, while the indoor CO<sub>2</sub> mixing ratio started increasing from about 420 ppm at 07:00 and peaked at about 600 ppm at 15:30. As expected, human exhaled breath is the major source of CO<sub>2</sub> in this office environment. However, monitoring indoor CO<sub>2</sub> remains critical in environments where other CO<sub>2</sub> sources exist, such as in indoor swimming pool facilities (e.g. CO<sub>2</sub> addition for maintaining water pH balance) and cooking with combustion sources (e.g. gas stoves).

The automated multi-point sampling system enables further examination of the spatial distribution of O<sub>3</sub> and CO<sub>2</sub> mixing ratios throughout the HVAC system. The O<sub>3</sub> mixing ratios along the supply air duct (sampled at Pre-Filter, Post-Filter, After HF, and SMSA) were relatively consistent, whereas the difference between SMSA and SMRA is markedly

greater. This indicates that the contribution of the HVAC filter bank and supply air duct surfaces to the loss of O<sub>3</sub> is less than that for the office space itself (see discussion in Section 3.3). The spatial distribution of O<sub>3</sub> and CO<sub>2</sub> at OA, RA (SMRA and PHRA), and SMSA are strongly connected with the HVAC system operational mode. Comparing indoor O<sub>3</sub> and CO<sub>2</sub> mixing ratios under the three ventilation modes, mode 3 with the highest outdoor AER resulted in the highest SMRA O<sub>3</sub> mixing ratio and the lowest SMRA CO<sub>2</sub> mixing ratio. At low outdoor AERs (Fig. 3(a.) and 3(b.)), indoor O<sub>3</sub> and CO<sub>2</sub> mixing ratios peaked at approximately 14 ppb and 610 ppm, respectively. When outdoor AERs were raised to medium (Fig. 3(c.) and 3(d.)) and high levels (Fig. 3(e.) and 3(f.)), the peak indoor O<sub>3</sub> mixing ratios increased to about 25 and 34 ppb, while the peak indoor CO<sub>2</sub> mixing ratios dropped to around 510 and 490 ppm, respectively.

Similarly, under the three pressurized ventilation operational modes 4 to 6, spatiotemporal trends were observed across the seven sampling locations (CA not shown) throughout the HVAC system (Fig. 4). The SMRA O<sub>3</sub> mixing ratios peaked at 16, 29, and 41 ppb under ventilation modes 4 to 6 with low, medium, and high outdoor AERs, respectively. The indoor O<sub>3</sub> mixing ratios under the pressurized ventilation modes were generally higher than those under the non-pressurized ventilation modes mainly due to the bias of the sampling periods – the pressurized ventilation modes were implemented in May and June 2019, during which the outdoor O<sub>3</sub> mixing ratios were also higher. With lower outdoor O<sub>3</sub> mixing ratios, the modeled indoor O<sub>3</sub> ratios under the

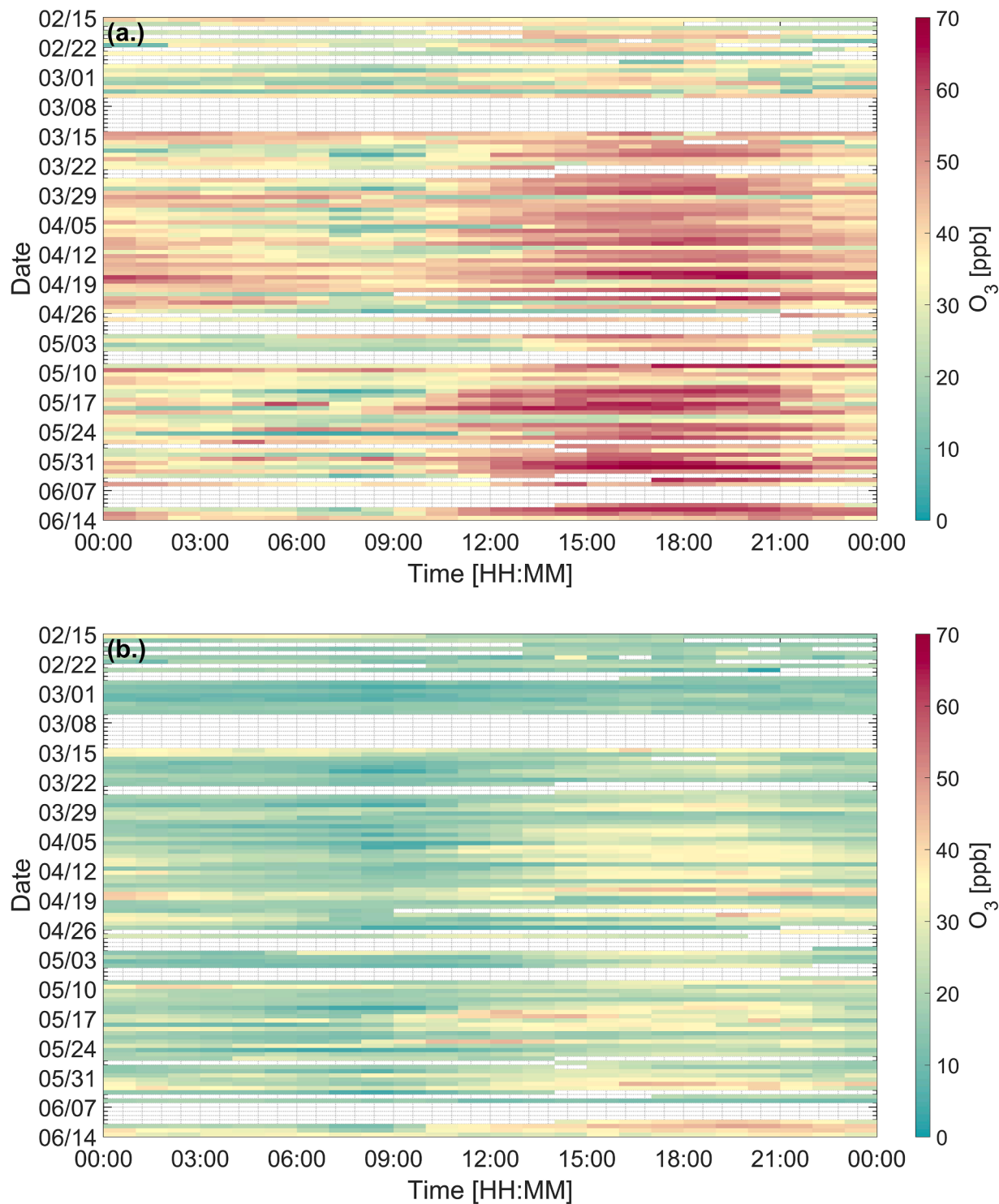


Fig. 5. Seasonal variation of (a.) outdoor O<sub>3</sub> mixing ratios and (b.) indoor O<sub>3</sub> mixing ratios. The blank areas indicate when measurement data was not available.

pressurized ventilation modes dropped as compared to their original levels (Fig. S1). Comparing mode 1 and mode 4 – low outdoor AER without and with pressurization – the outdoor O<sub>3</sub> mixing ratios under mode 4 were about 1.5 to 2 times higher than the outdoor O<sub>3</sub> mixing ratios under mode 1, while the indoor O<sub>3</sub> mixing ratios under the two modes were around the same levels. This indicates that pressurizing the room can effectively prevent outdoor air pollutants from infiltrating into indoor environments.

Fig. 5 shows the diurnal and seasonal variations of indoor and outdoor O<sub>3</sub> mixing ratios during the measurement campaign. The ranges for

diurnal outdoor O<sub>3</sub> mixing ratios increased from 10 to 30 ppb in February/March to 20 to 60 ppb in June with the increase in the duration of daylight. The trend observed in this study agrees with the seasonal O<sub>3</sub> variations observed in the Midwest region of the U.S. [57]. The median indoor O<sub>3</sub> mixing ratios during the campaign varied from 15 to 25 ppb throughout the day, while the median outdoor O<sub>3</sub> mixing ratios varied from 15 to 60 ppb (Fig. 6). This suggests that the building operation strategy for maintaining acceptable IAQ cannot be simply based on CO<sub>2</sub> mixing ratios. For instance, when the occupancy level is high, the typical control strategy is to increase the outdoor AER.



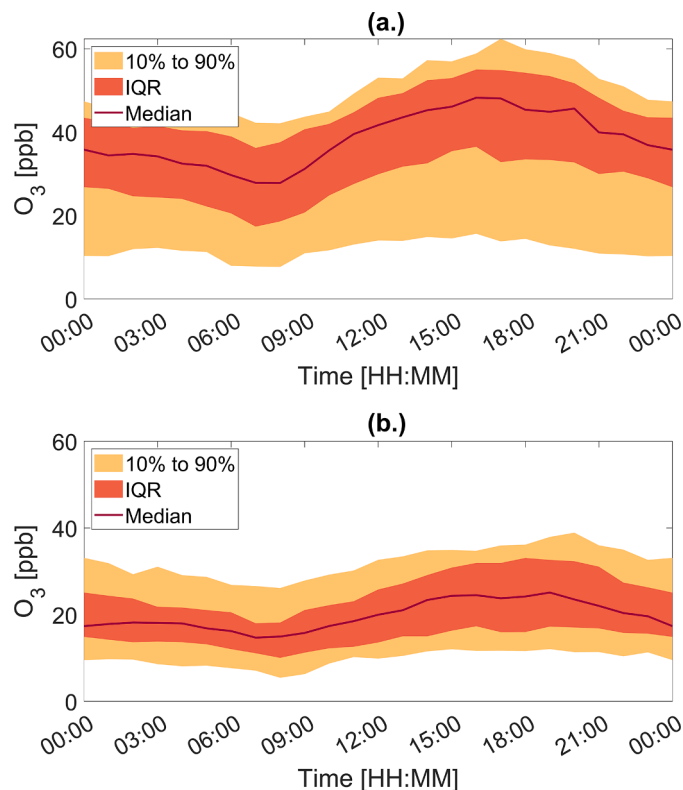


Fig. 6. Median, interquartile range, and 10th- to 90th-percentile of (a.) outdoor  $O_3$  mixing ratios and (b.) indoor  $O_3$  mixing ratios.

However, introducing more outdoor air to dilute indoor-generated  $CO_2$  in the summer can lead to significant increases in indoor  $O_3$  mixing ratios due to the high outdoor  $O_3$  mixing ratios. With high occupancy levels and elevated  $O_3$  mixing ratios, it is more likely that  $O_3$  reactions with human skin lipids and body surfaces would form more secondary products that may potentially lead to adverse health effects. Thus, it is critical to consider the combined effects of different trace gases on IAQ when controlling a building ventilation system.

### 3.2. $O_3$ and $CO_2$ I/O ratios under different outdoor AERs and human occupancy levels

Spatiotemporal sampling of  $O_3$  and  $CO_2$  under different HVAC system operational modes revealed relationships between human occupancy, outdoors AERs, and  $O_3$  and  $CO_2$  mixing ratios. Given that outdoor  $O_3$  is the major source of indoor  $O_3$  in the office environment, the dependence of indoor  $O_3$  mixing ratios on the outdoor AER and occupancy could be biased by the outdoor  $O_3$  concentration. The indoor/outdoor (I/O) ratio normalizes indoor mixing ratios by outdoor mixing ratios, providing a more unbiased term to examine the contribution of indoor- and outdoor-associated sources of trace gases. Fig. 7 groups the I/O ratios of  $O_3$  and  $CO_2$  mixing ratios by outdoor AERs and office occupancy levels. The occupancy levels were categorized as: (1.) low occupancy where occupancy < 4; (2.) medium occupancy where  $4 \leq$  occupancy < 8; and (3.) high occupancy where occupancy  $\geq$  8.

Generally, the I/O ratio of  $O_3$  increases with an increase in the outdoor AER (Fig. 7(a.)). Median  $O_3$  I/O ratios ranged from 0.35 to 0.43, 0.45 to 0.55, and 0.60 to 0.65 at low, medium, and high AERs, respectively.  $O_3$  I/O ratios observed in this study are consistent with the I/O ratio reported for an office in the U.S. with similar AERs [60], but higher than those reported in offices in Europe and Asia where AERs were not specified [61,62]. The dependency of the  $O_3$  I/O ratio on occupancy levels varies with outdoor AERs. The median  $O_3$  I/O ratio decreased by 11 to 22% from low/medium occupancy to high occupancy levels at low

outdoor AERs, and by 4 to 22% at medium outdoor AERs. Interestingly, at high outdoor AERs, the median  $O_3$  I/O ratios were around the same level as at low and medium occupancy, and only elevated by 8% from low/medium to high occupancy levels. At lower AERs, air exchange due to ventilation contributed less to the loss mechanisms of indoor  $O_3$ , while  $O_3$  deposition to human body surfaces contributed more to the loss of indoor  $O_3$ . Thus, with higher occupancy, lower I/O ratios can be observed, as the increase of  $O_3$  deposition leads to lower indoor  $O_3$  mixing ratios. In contrast, at higher AERs, outdoor air ventilation via the HVAC system became the major sink for indoor  $O_3$  and deposition to occupant surfaces contributed less to indoor  $O_3$  loss, so the  $O_3$  I/O ratio can be less dependent on occupancy levels at higher AERs. An ANOVA analysis (Table 4) also verified the dependence of the  $O_3$  I/O ratio on occupancy and AER ( $p < 0.001$ ).

Fig. 7(b.) presents the “ $O_3$  loss” at different AERs and occupancy levels. The median value of the  $O_3$  loss at different outdoor AERs and occupancy levels varies from 14.7 to 26.5 ppb, similar to the  $O_3$  loss ranges modeled by Weschler and Nazaroff (15.5 to 27.8 ppb) for different indoor environments with AERs from 0.25 to  $6 \text{ h}^{-1}$  and occupancy levels from 2 to 35 occupants [56]. At low and medium AERs, the  $O_3$  loss increased with the increase in occupancy levels due to deposition onto human body surfaces. This indicates more secondary products can be formed indoors with higher occupancy, which may lead to adverse health effects. The  $O_3$  loss decreased when the outdoor AER increased, as higher AER leads to elevated indoor  $O_3$  mixing ratios. This suggests that HVAC system control and building ventilation strategies need to be adjusted accordingly based on the outdoor AER and occupancy levels.

Fig. 7(c.) illustrates the dependency of the  $CO_2$  I/O ratio on the AER and occupancy levels. Occupants are the major source of indoor  $CO_2$  and the median  $CO_2$  I/O ratios increased with the increase of occupancy levels, increasing from 1.01 to 1.03 (low occupancy) to 1.22 to 1.45 (high occupancy) at various AERs. However, the increase in the outdoor AER did not always lower the  $CO_2$  I/O ratio. At low and medium occupancy (less than 8 occupants), the median  $CO_2$  I/O ratios were similar across low, medium, and high AERs. The consistency in  $CO_2$  I/O ratios across different AERs indicates that with lower occupancy, a low AER would be sufficient to control the indoor  $CO_2$  mixing ratio at low levels, with an I/O ratio very close to 1. At high occupancy levels (> 8 occupants), the  $CO_2$  I/O ratio dropped by 5% and 15% when the AER increased from low to medium and high levels, respectively. Increasing outdoor AERs can effectively dilute indoor-generated  $CO_2$  at higher occupancy levels. Fig. 7 demonstrates that the outdoor AERs and occupancy levels may have different effects on  $O_3$  and  $CO_2$  I/O ratios. Building ventilation strategies may need to be implemented with careful and comprehensive consideration of their impact on different indoor air pollutants. Recent studies have evaluated the performance of low-cost metal oxide and electrochemical sensors for trace gases, including  $O_3$ , CO, and  $NO_x$  [63–65]. With proper calibration, these low-cost sensors showed good correlation with reference analytical instruments. A multi-point trace gas sampling system such as that presented in this study, or implemented through the use of low-cost sensor arrays, has the potential to be integrated into building automation systems to aid the ventilation control of buildings and improve IAQ.

### 3.3. $O_3$ loss in the office and its HVAC system

Fig. 8 illustrates diurnal trends of overall and itemized  $O_3$  loss rates during representative days for three different ventilation operational modes – 1, 2, and 3. The loss rate due to  $O_3$  deposition to human body surfaces generally follows the trend of indoor occupancy levels, indicating that the occupancy level is strongly connected to indoor  $O_3$  loss mechanisms and indoor  $O_3$  mixing ratios. For the ventilation mode 1 with the lowest AER, the overall  $O_3$  loss rate ranges from 5.3 to  $9.2 \text{ h}^{-1}$  throughout the day. This variation is mainly due to the variation in  $O_3$  deposition to human body surfaces, which ranges from 0 to  $2.3 \text{ h}^{-1}$  with occupancy ranging from 0 to 9.5.  $O_3$  loss due to ventilation and

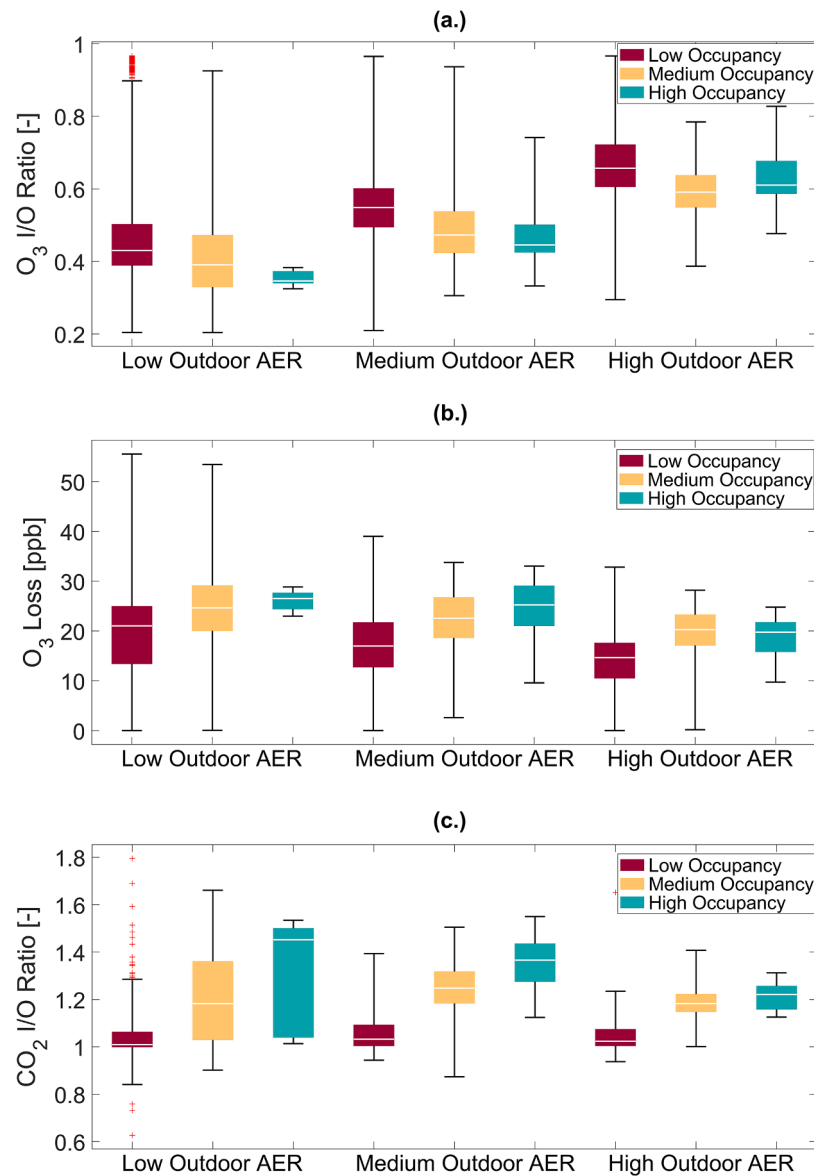


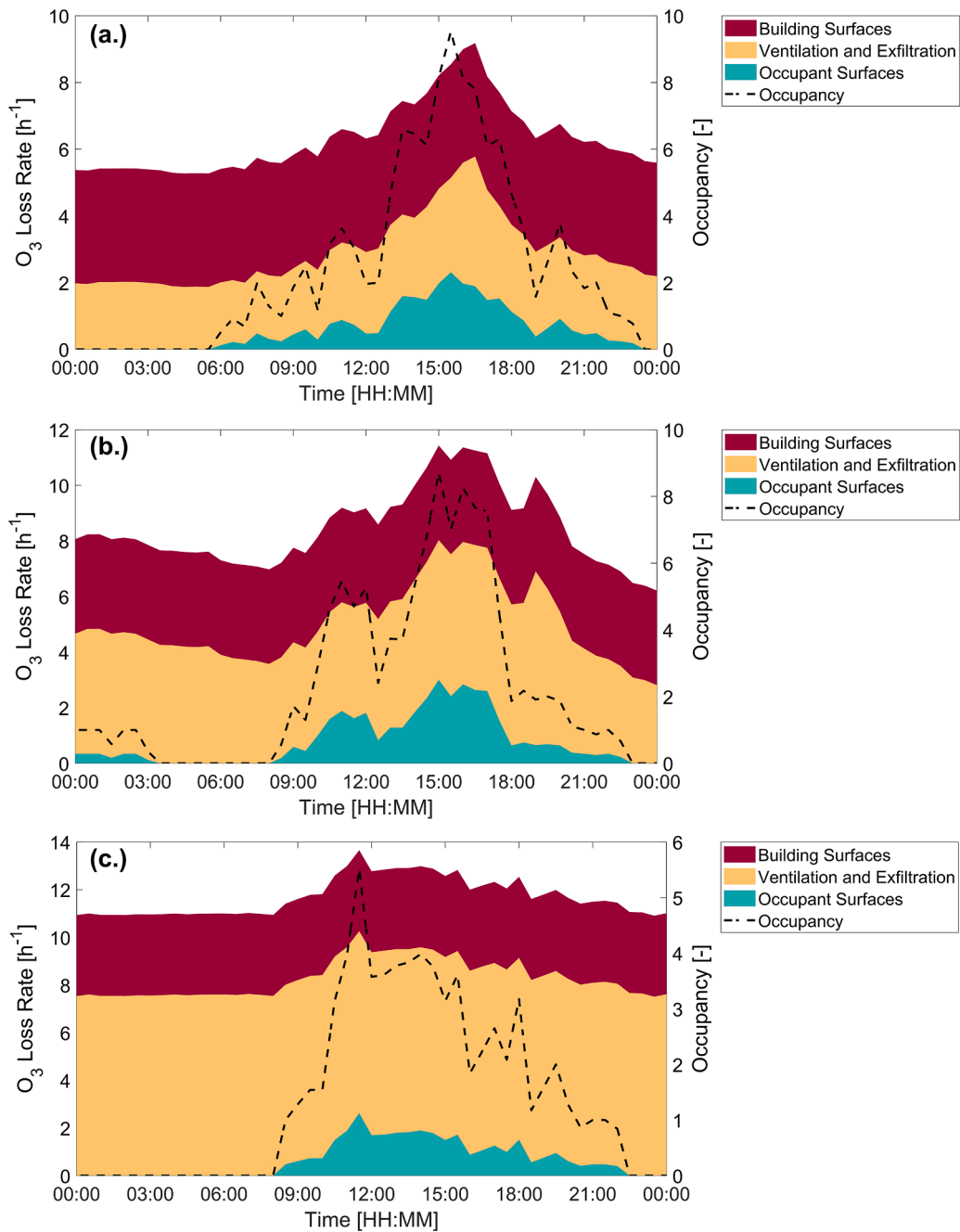
Fig. 7. (a.) I/O ratios of O<sub>3</sub> mixing ratios, (b.) O<sub>3</sub> loss, and (c.) I/O ratios of CO<sub>2</sub> mixing ratios for different outdoor AERs and office occupancy levels during the field measurement campaign.

**Table 4**  
ANOVA test for the dependence of the O<sub>3</sub> I/O ratio on AER and occupancy.

	Sum of Squares	DF	Sum of Squares	F	p-Value
AER	19.007	3	6.3357	394.82	1.46E-226
Occupancy	2.5740	2	1.2870	80.203	6.33E-35
Error	69.114	4307	0.016047		
Total	90.296	4312			

exfiltration dominated the overall O<sub>3</sub> loss rate under all three ventilation modes for our study site, an office environment without indoor combustion sources (e.g. gas stoves). Outdoor air ventilation and exfiltration, deposition to building material surfaces, and deposition to human body surfaces account for 37 to 64%, 36 to 37%, and 0 to 26% of the overall O<sub>3</sub> loss rate, respectively. The contribution of gas-phase reactions to the overall O<sub>3</sub> loss rate was negligible. It should be noted that in other indoor environments, especially those with combustion sources and the use of personal care products, O<sub>3</sub> reactions with NO<sub>x</sub> and VOCs could be important sinks. The overall O<sub>3</sub> loss rates were the highest,

approaching a peak of around 14 h<sup>-1</sup>, under ventilation mode 3 with the highest outdoor AER. Outdoor air ventilation contributed 44 to 51% to the overall O<sub>3</sub> loss rate under mode 2 and 56 to 69% to the overall O<sub>3</sub> loss rate under mode 3. For O<sub>3</sub> deposition to human body surfaces, the loss rate peaked at 3.0 h<sup>-1</sup> and 2.6 h<sup>-1</sup> when the office occupancy level peaked at 8.7 and 5.5 under ventilation modes 2 and 3, respectively. Average O<sub>3</sub> deposition velocities to human body surfaces during the three days selected from ventilation modes 1 to 3 were estimated to be 48, 68, and 93 m h<sup>-1</sup>, which results in O<sub>3</sub> loss rates of 0.25, 0.35, and 0.47 h<sup>-1</sup> person<sup>-1</sup>, respectively. The deposition velocities to human body surfaces observed in this study were 2 to 10 times higher than the values reported in previous studies [12,16,66–68]. O<sub>3</sub> loss rates reported in these studies vary from 0.02 h<sup>-1</sup> person<sup>-1</sup> to 1.5 h<sup>-1</sup> person<sup>-1</sup>, with room volumes ranging from 22.5 to 670 m<sup>3</sup>. O<sub>3</sub> loss rates observed in our study were 1.5 to 4.7 times higher than the O<sub>3</sub> loss rates for a simulated office with a similar room volume [69]. Such inconsistencies might be explained by differences in the application of personal care products to the skin and hair [70] or the clothing the occupants were wearing [13, 71], especially for mode 3 which occurred in late April. The deposition velocities can be enhanced due to O<sub>3</sub> reactions with hair (large surface



**Fig. 8.** Diurnal trends of the overall and itemized O<sub>3</sub> loss rates (left axis) and occupancy (right axis) during representative days for three different ventilation modes: (a.) ventilation mode 1, (b.) ventilation mode 2, and (c.) ventilation mode 3.

area), soiled clothing (multiple layers may be worn), and soiled backpacks. Thus, the assumed occupant surface area might be underestimated. Time delays in indoor and outdoor O<sub>3</sub> sampling and the time-averaged loss rate calculation may also lead to uncertainties in the estimation of the deposition velocities.

Considering the large surface area of the HVAC system and its components, the HVAC system can be another important sink of outdoor O<sub>3</sub> being supplied to indoor environments. The multi-point sampling system allows for the investigation of how each HVAC component can impact O<sub>3</sub> dynamics. Fig. 9(a.) presents the diurnal trend of O<sub>3</sub> loss across the HVAC filter bank during the pre/post-filter sequence. Pre-Filter O<sub>3</sub> mixing ratios were consistently higher than the Post-Filter mixing ratios, indicating O<sub>3</sub> deposition to the filter media. Up to 8 ppb of O<sub>3</sub> could deposit to the HVAC filter bank when passing through,

resulting in 0 to 10% of O<sub>3</sub> loss across the filter. O<sub>3</sub> removal by other HVAC system components is presented in Fig. 9(b.). O<sub>3</sub> mixing ratios dropped by a median of 3.9, 1.2, 0.5, and 6.0% when travelling from Pre-Filter to SMSA, Post-Filter to SMSA, After HF to SMSA, and SMRA to PHRA, respectively. The difference between O<sub>3</sub> removal percentages from Pre-Filter and Post-Filter to SMSA verifies the contribution of the HVAC filter bank to O<sub>3</sub> deposition. For the other parts of the HVAC system, the HVAC duct itself could be major contributor of O<sub>3</sub> loss. The HVAC duct between SMRA and PHRA was the longest among the four segments, with only one return fan installed along the duct. Both sampling locations were located along the return air duct, however, up to 18% of O<sub>3</sub> loss was observed. With the two sampling locations at two different mechanical rooms that are > 10 m apart, the large surface area of the long HVAC duct contributes to the high O<sub>3</sub> loss from SMRA to



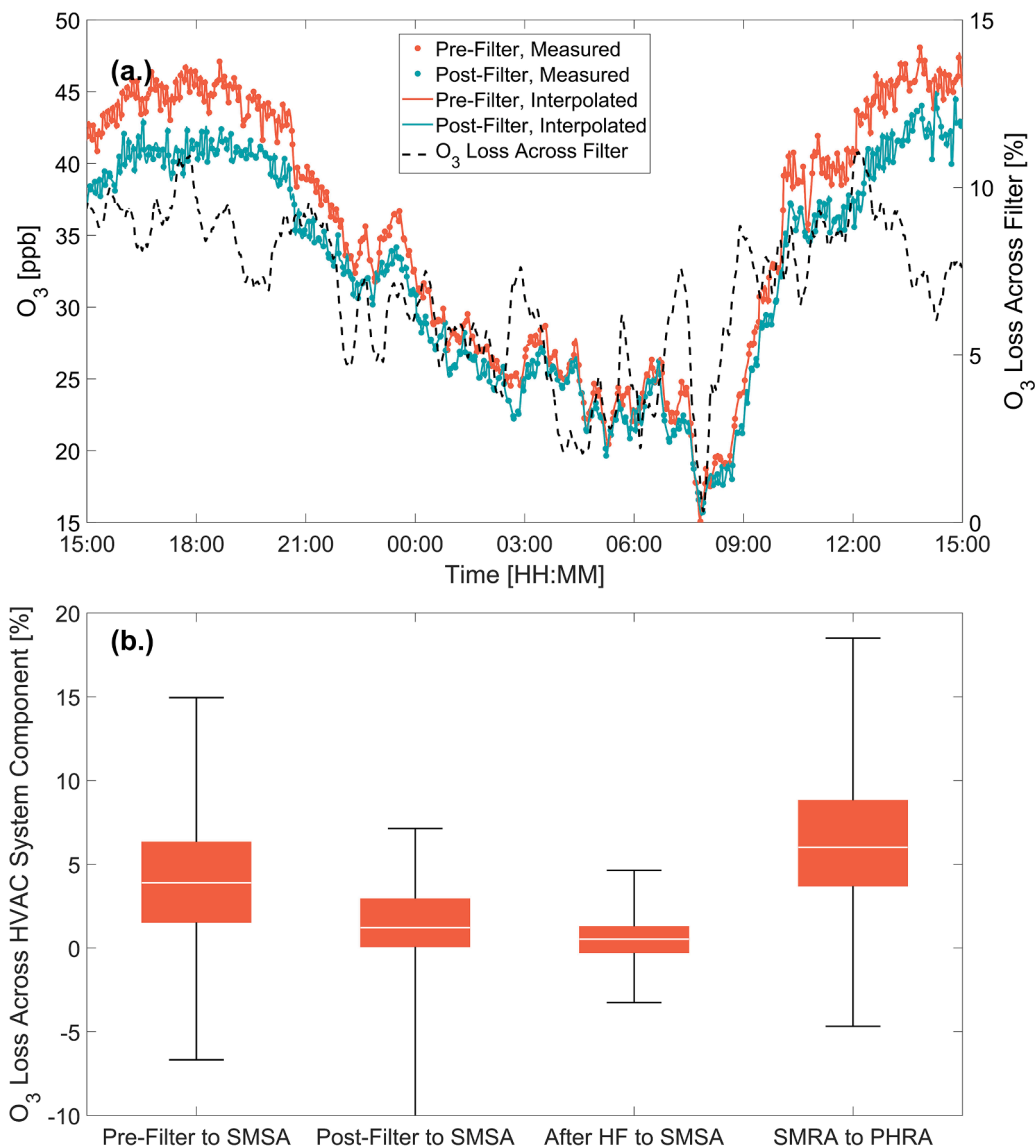


Fig. 9. (a.) Diurnal trend of the O<sub>3</sub> loss across the HVAC filter bank during the pre-/post-filter sampling sequence and (b.) O<sub>3</sub> loss across different HVAC system components during the field measurement campaign.

PHRA.

It is important to note that the findings presented here may be specific to the office environment studied. In office settings, as demonstrated in this study, occupants are major contributors to O<sub>3</sub> loss, with large-scale and complex HVAC systems also playing an important role in the removal of O<sub>3</sub>. In contrast, in residential environments with lower occupancy and simpler HVAC systems, the contribution of occupants and HVAC systems to O<sub>3</sub> loss may be less significant. However, other indoor activities – such as cooking with gas stoves, which emit NO<sub>x</sub>, or the use of personal care products that release VOCs – can lead to O<sub>3</sub> reactions with these compounds and the production of secondary pollutants. These processes may become the dominant mechanisms for O<sub>3</sub> loss in such environments. This highlights the need for multi-point sampling of trace gases across diverse indoor environments and their associated HVAC systems to fully understand the fate and dynamics of indoor air pollutants.

4. Conclusions and future directions

This study introduced a novel automated multi-point sampling system for the real-time monitoring of spatiotemporal trends of trace gases

in the HVAC system of a LEED-certified office building. A four-month field measurement campaign at the Purdue University Ray W. Herrick Laboratories provided valuable insights into the dynamics of O<sub>3</sub> and CO<sub>2</sub> within the HVAC system under varying ventilation modes and occupancy levels. Through O<sub>3</sub> and CO<sub>2</sub> measurements with the multi-point sampling system at eight different locations throughout the HVAC system in a LEED-certified office building, spatiotemporal variations in O<sub>3</sub> and CO<sub>2</sub> mixing ratios were observed. The results revealed significant diurnal and seasonal variations in both indoor and outdoor O<sub>3</sub> mixing ratios, emphasizing the influence of outdoor air on IAQ. Higher outdoor AERs were associated with increased indoor O<sub>3</sub> levels but reduced CO<sub>2</sub> concentrations, demonstrating the complex interplay between ventilation rates and indoor air pollutant levels. Spatial variations of O<sub>3</sub> and CO<sub>2</sub> mixing ratios were observed as well, especially across supply air, return air, and outdoor air sampling locations. Minor variations were also observed among different locations along the supply duct (Pre-Filter, Post-Filter, After-HF, and SMSA) and return duct (SMRA and PHRA), indicating that the HVAC filter bank and duct surfaces can contribute to O<sub>3</sub> loss.

I/O ratios of O<sub>3</sub> and CO<sub>2</sub> ranged from 0.35 to 0.65 and 1.01 to 1.45, respectively, at different outdoor AERs and occupancy levels.

Ventilation and occupancy impact the I/O ratio of O<sub>3</sub> and CO<sub>2</sub> differently, which needs to be carefully accounted for when implementing building ventilation strategies to improve IAQ. Overall O<sub>3</sub> loss rates and the contribution of each loss mechanism were estimated, including building ventilation and exfiltration (outdoor air exchange), gas-phase O<sub>3</sub> reactions with NO<sub>x</sub> and selected VOCs, and O<sub>3</sub> deposition to building material surfaces and human body surfaces. Ventilation and exfiltration contributed most to the overall O<sub>3</sub> loss rate, and the contribution of deposition on human body surfaces cannot be neglected when the occupancy level is high. Human occupancy significantly influenced O<sub>3</sub> deposition onto skin and clothing, further complicating the dynamics of IAQ. This study also highlighted the role of HVAC components, such as filters and ducts, in contributing to O<sub>3</sub> loss, with up to 18% O<sub>3</sub> removal observed in the longest HVAC duct segment.

Current ventilation standards, which primarily consider CO<sub>2</sub> mixing ratios in the design of the ventilation rate, may not adequately address the complexities of indoor air composition and the interactions between various pollutants. This study emphasizes the need for a more comprehensive IAQ sensing system and ventilation control strategies that consider multiple trace gases to effectively optimize IAQ. The automated multi-point sampling system presented in this study has the potential to be implemented in building automation systems to aid in the control of building ventilation. This system can be connected not only to CO<sub>2</sub> and O<sub>3</sub> analyzers as demonstrated in this study, but also to advanced analytical instruments, including online mass spectrometers such as PTR-TOF-MS, which can characterize a wide range of VOCs. While installing individual instruments at each sampling location would be impractical due to their size, cost, and calibration needs, a multi-point sampling system with an automatic multi-port valve offers a practical solution for precise spatiotemporal trace gas monitoring. By integrating this system into building automation systems, real-time data of multiple trace gases (e.g. CO<sub>2</sub>, O<sub>3</sub>, NO<sub>x</sub>, VOCs) can be utilized to develop dynamic ventilation strategies that respond to varying indoor and outdoor conditions. For example, by measuring trace gases at the supply, return, and outdoor air ducts, building automation systems can better understand the sources, transport, and fate of these air pollutants.

Real-time monitoring of the spatial distribution of trace gases at these locations can be further inputted into indoor chemistry models to predict the concentration of these pollutants and their possible secondary reaction products in order to find optimal settings for the HVAC system. This dynamic approach helps maintain a healthy indoor environment while minimizing energy consumption by avoiding unnecessary ventilation during periods of high outdoor air pollution. In addition, by arranging the sampling locations upstream and downstream of in-duct air filtration and purification devices (such as HVAC filter banks, UV disinfection units, or bi-polar ionization devices), the system can evaluate the in-situ performance of these technologies, ensuring that these devices are functioning properly and informing when maintenance is needed. We believe that the innovative integration of automated multi-point sampling, analytical instrumentation, and building automation systems opens up new possibilities for real-time air quality monitoring and predicting and enabling more accurate and efficient HVAC system operation and maintenance across diverse indoor environments. Such systems have the potential to help reduce building maintenance and operational costs while improving the health and productivity of office workers. Future research should focus on exploring their integration with other smart building technologies, optimizing the control of HVAC systems, analyzing their cost benefits, and evaluating their long-term benefits in diverse building types and climates.

#### CRediT authorship contribution statement

**Jinglin Jiang:** Writing – review & editing, Writing – original draft, Visualization, Validation, Methodology, Investigation, Formal analysis, Data curation. **Junkai Huang:** Visualization, Methodology, Formal

analysis. **Nusrat Jung:** Writing – review & editing, Writing – original draft, Visualization, Validation, Supervision, Software, Resources, Project administration, Methodology, Investigation, Funding acquisition, Formal analysis, Data curation, Conceptualization. **Brandon E. Boor:** Writing – review & editing, Writing – original draft, Visualization, Validation, Supervision, Software, Resources, Project administration, Methodology, Investigation, Funding acquisition, Formal analysis, Data curation, Conceptualization.

#### Declaration of competing interest

The authors declare that they have no conflict of interest.

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#### Supplementary materials

Supplementary material associated with this article can be found, in the online version, at [doi:10.1016/j.buildenv.2025.112651](https://doi.org/10.1016/j.buildenv.2025.112651).

#### Data availability

Data will be made available on request.

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